



# DeepWind-An Innovative Offshore Wind Turbine Concept

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Innovative Concepts and new Technologies

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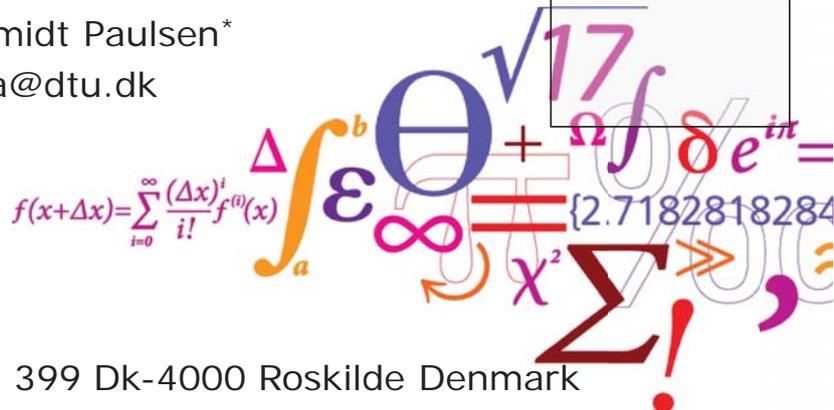


EU FP7+ DeepWind Consortium  
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**DTU Wind Energy**  
Department of Wind Energy

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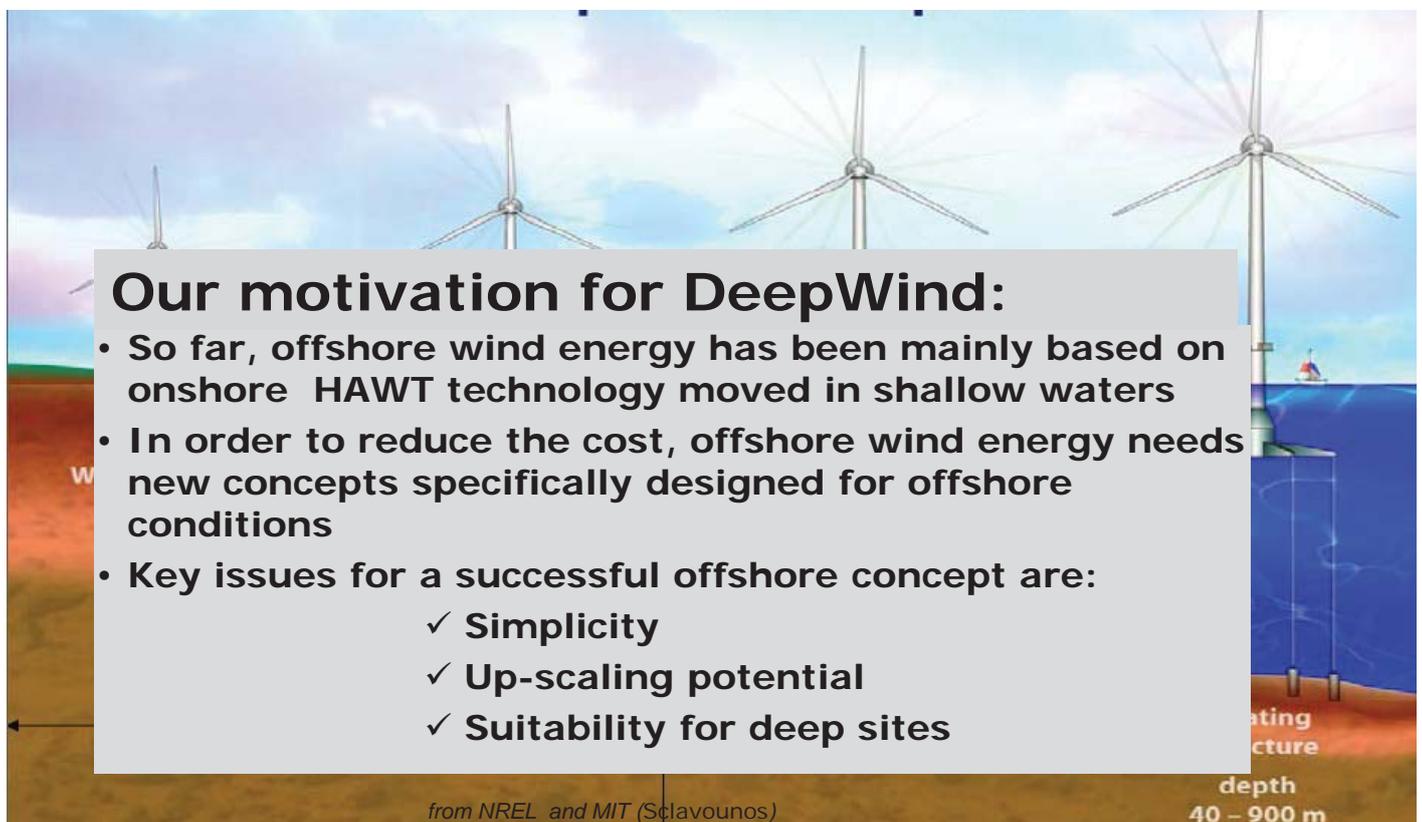
# DeepWind

## Contents

- DeepWind Concept
- DeepWind instruments and goals
- Results in the project
- Conclusion

DeepWind

## Concept From shore to deep sea



## DeepWind

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# DeepWind

## DeepWind Concept

- Genuine offshore concept
  - Simple
  - Scalable
  - For deep sea sites
- Floating offshore based on VAWT technology
- Cost difference allows room for design space
- A priori: Tech Range 100-1000 m depth
- So far
  - demonstrator
  - exhibition model
  - paper work results
- Contributions to reduce risk(selected list):

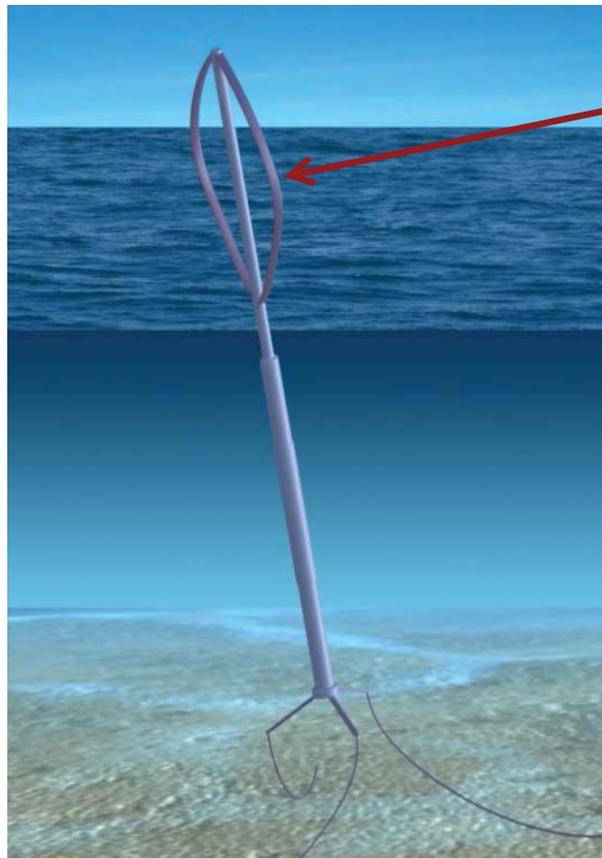


- Vita L, Paulsen US, Pedersen TF, Madsen HA, Rasmussen F *A Novel Floating Offshore Wind Turbine Concept* in Proceedings of the European Wind Energy Conference (EWEC), Marseille, France, 2009.
- Vita L, Zhale F, Paulsen US Pedersen TF, Madsen HA, Rasmussen F. *Novel Concept For Floating Offshore Wind Turbines: Concept Description And Investigation Of Lift, Drag And Friction Acting On The Rotating Foundation* in Proceedings of the ASME 2010 29<sup>th</sup> International Conference on Ocean, Offshore and Arctic Engineering, June 6 Shanghai 2010 OMAE2010-20357
- Vita L *Offshore floating vertical axis wind turbines with rotating platform* Risø DTU, Roskilde, Denmark, PhD dissertation PhD 80, 2011
- Stefan Carstensen<sup>1</sup> Xerxes Mandviwalla, Luca Vita and Uwe Schmidt Paulsen *Lift of a Rotating Circular Cylinder in Unsteady Flows* ISOPE June 2012

# DeepWind

## The Concept

- No pitch, no yaw system

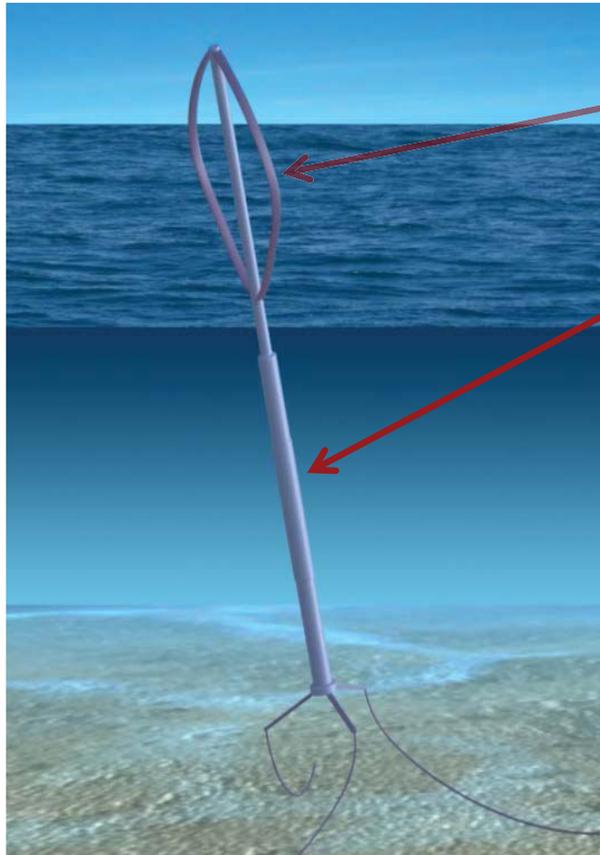


- Light weight rotor with pultruded blades

# DeepWind

## The Concept

- No pitch, no yaw system
- Floating and rotating tube as a spar buoy

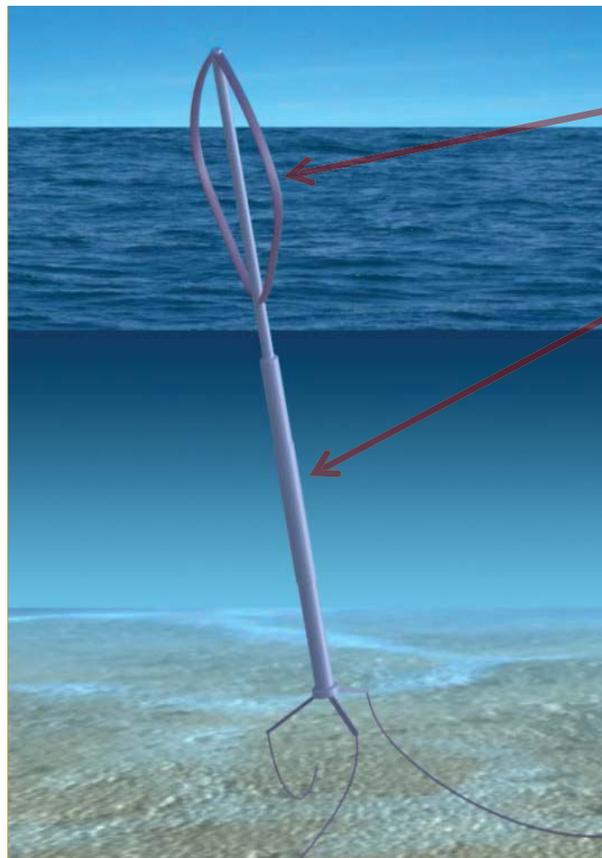


- Light weight rotor with pultruded blades
- Long slender and rotating underwater tube with little friction

# DeepWind

## The Concept

- No pitch, no yaw system
- Floating and rotating tube as a spar buoy
- C.O.G. very low – counter weight at bottom of tube

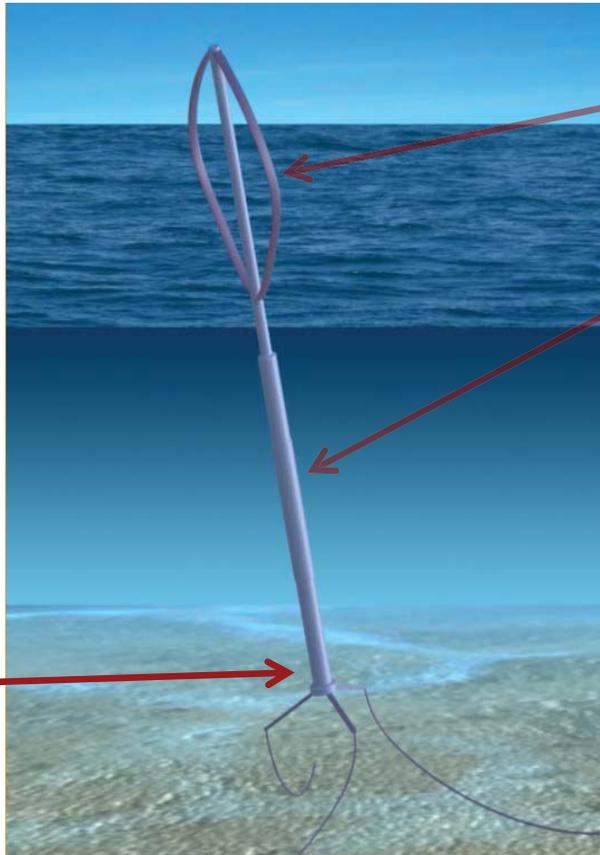


- Light weight rotor with pultruded blades
- Long slender and rotating underwater tube with little friction with little friction

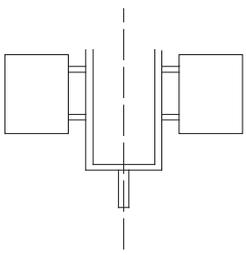
# DeepWind

## The Concept

- No pitch, no yaw system
- Floating and rotating tube as a spar buoy
- C.O.G. very low – counter weight at bottom of tube
- Safety system



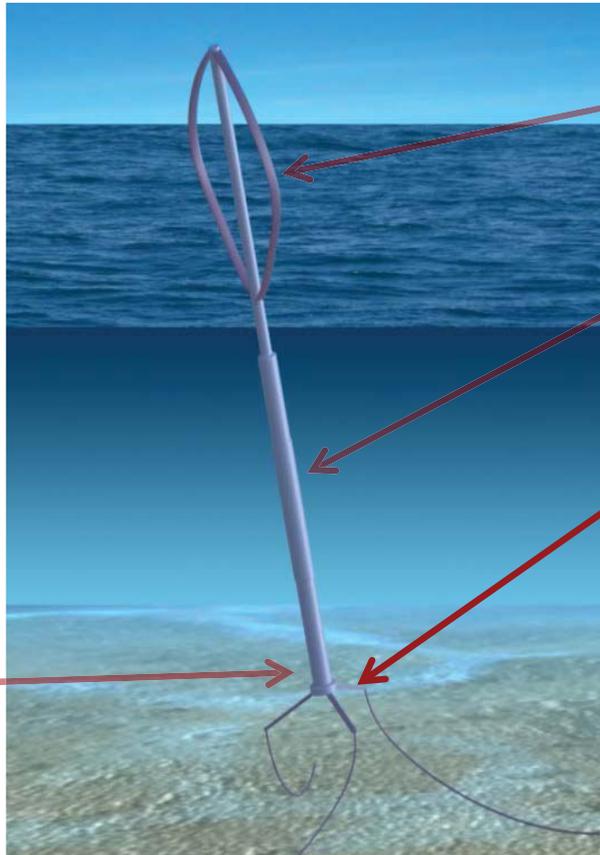
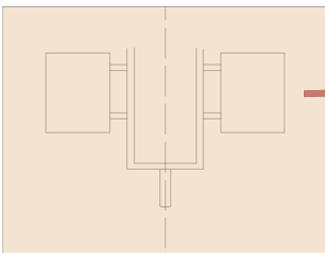
- Light weight rotor with pultruded blades
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# DeepWind

## The Concept

- No pitch, no yaw system
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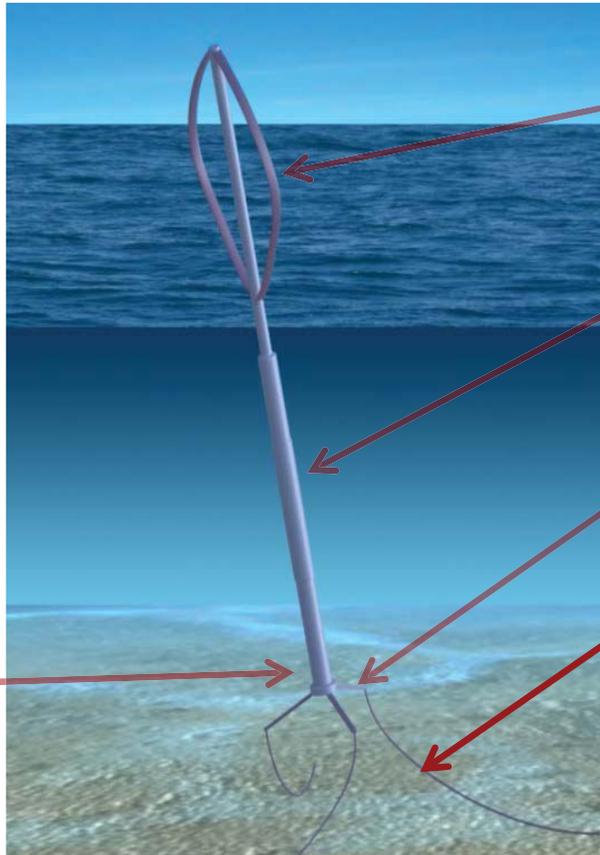
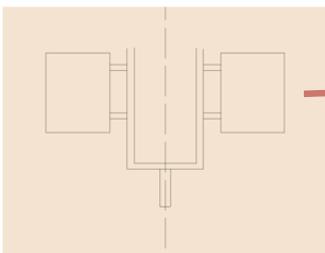


- Light weight rotor with pultruded blades
- Long slender and rotating underwater tube with little friction
- Torque absorption system

# DeepWind

## The Concept

- No pitch, no yaw system
- Floating and rotating tube as a spar buoy
- C.O.G. very low – counter weight at bottom of tube
- Safety system

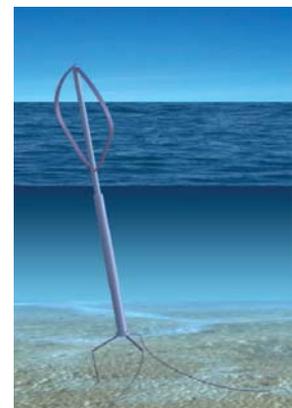
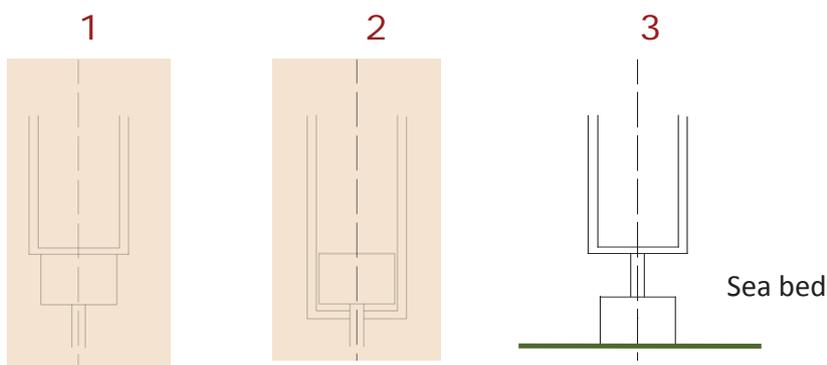


- Light weight rotor with pultruded blades
- Long slender and rotating underwater tube with little friction
- Torque absorption system
- Mooring system

## DeepWind

### The Concept- Generator configurations

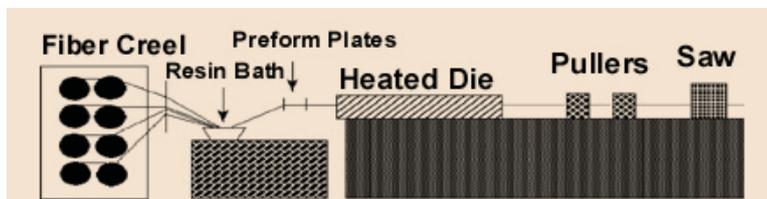
- The Generator is at the bottom end of the tube; several configurations are possible to convert the energy
- Three selected to be investigated first:
  1. Generator fixed on the torque arms, shaft rotating with the tower
  2. Generator inside the structure and rotating with the tower. Shaft fixed to the torque arms
  3. Generator fixed on the sea bed and tower. The tower is fixed on the bottom (not floating).



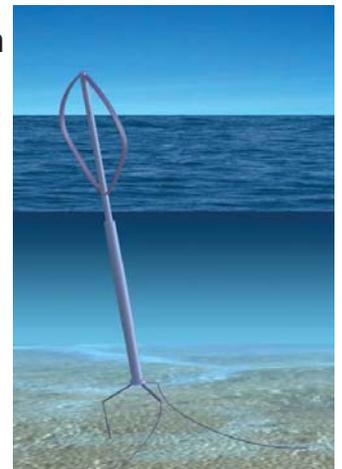
## DeepWind

# The Concept- Blades technology

- The blade geometry is constant along the blade length
- The blades can be produced in GRP or similar
- Pultrusion technology:
  - Presently block up to approx 1 m units
  - outlook- 11 m chord several 100 m long blade length



- Pultrusion technology could be performed on a ship at site
- Blades can be produced in modules



## DeepWind

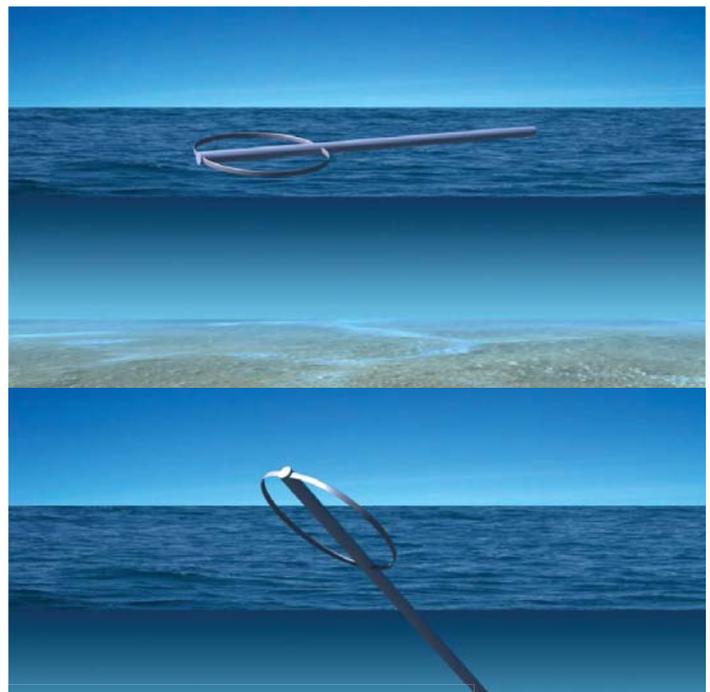
# The Concept- Installation, Operation & Maintenance

- INSTALLATION

- ✓ Using a two bladed rotor, the turbine and the rotor can be towed to the site by a ship. The structure, without counterweight, can float horizontally in the water. Ballast can be gradually added to tilt up the turbine.

- O&M

- ✓ Moving the counterweight in the bottom of the foundation is possible to tilt up the submerged part for service.
- ✓ It is possible to place a lift inside the tubular structure.



- A new basis for cost cutoff in installation procedures
- Redistributing the costs

## DeepWind

# The Concept Upscaling

- Pultrusion technology allows for very long and fail-free manufactured blades
- Concept simplicity
- Few components with less down time failures
- Cost-effective different materials for large structure
- Specific requirements to maintain the underwater components

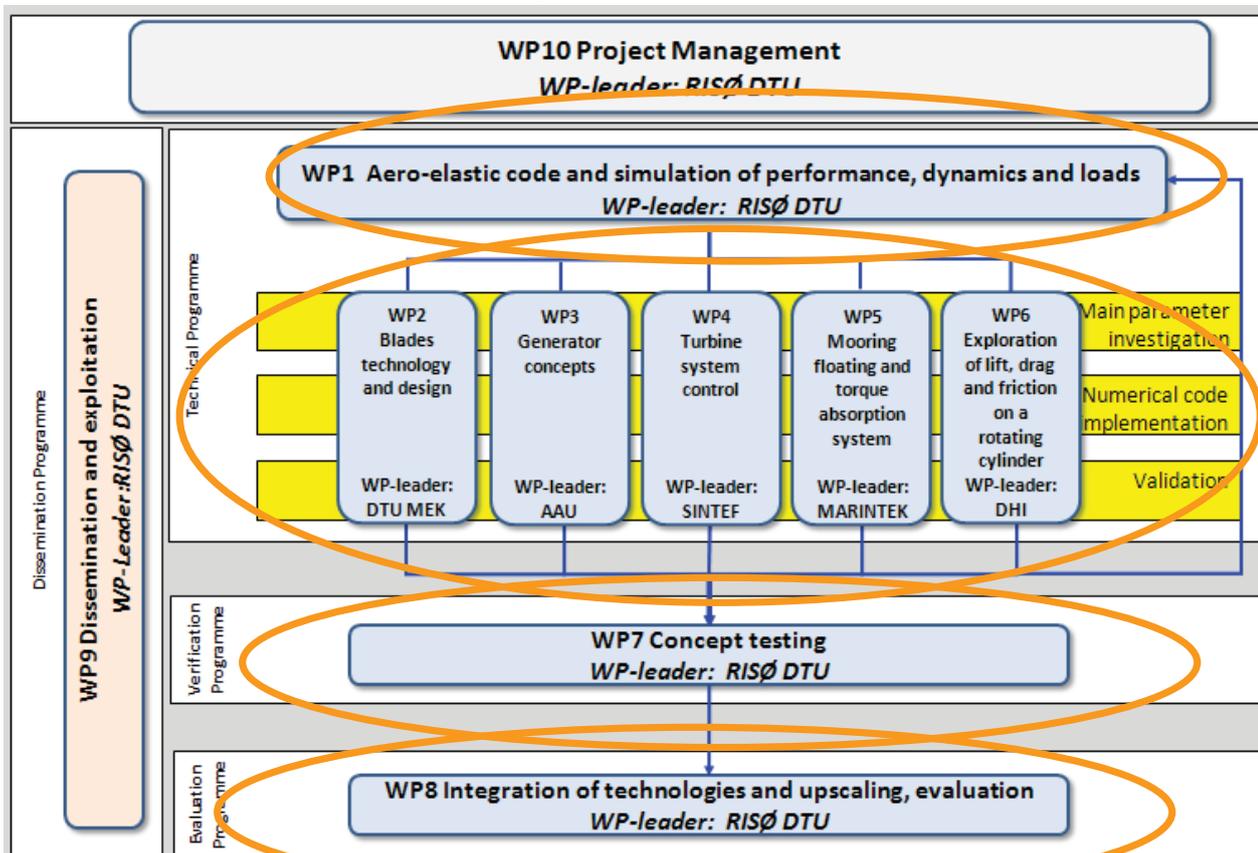


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# DeepWind Instruments and goals



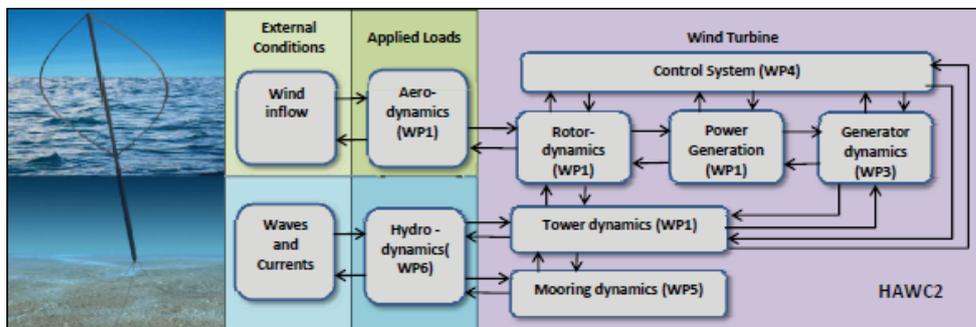
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# DeepWind

## Results 1st Design Assumptions



- Dynamic stall neglected
- Atmospheric turbulence not considered
- Evaluation of loads with 3 DOF
- No mooring

# DeepWind

## Results 2MW VAWT vs HAWT



	Deep Wind	HyWind*
<b>Power</b>	2 MW	2.3 MW (+15%)
<b>Rotor Diameter</b>	67 m	82.4 m (+23%)
<b>Rotor Height</b>	75 m	65.0 m (-13%)
<b>Chord (blades number)</b>	3.2 m (2)	N/A (3)
<b>Rotational speed at rated conditions</b>	15.0 rpm	16.0 rpm (+7%)
<b>Radius of the rotor tower</b>	2.0 m	3.0 m (+50%)
<b>Maximum radius of the submerged part</b>	3.5 m	4.15 m (+19%)
<b>Total tower length (underwater part)</b>	183 m (93m)	165 (100)
<b>Displacement</b>	3000 tons	5300 tons (+77%)



\*"HYWIND, Concept, challenges and opportunities ", Statoil

Uwe Schmidt Paulsen *Prospects of Large Floating Vertical Axis Wind Turbines* Proceedings in Deep Sea Offshore R&D Conference Trondheim(NO) 2011

# DeepWind

## Results 1<sup>st</sup> BaseLine 5 MW Rotor Design

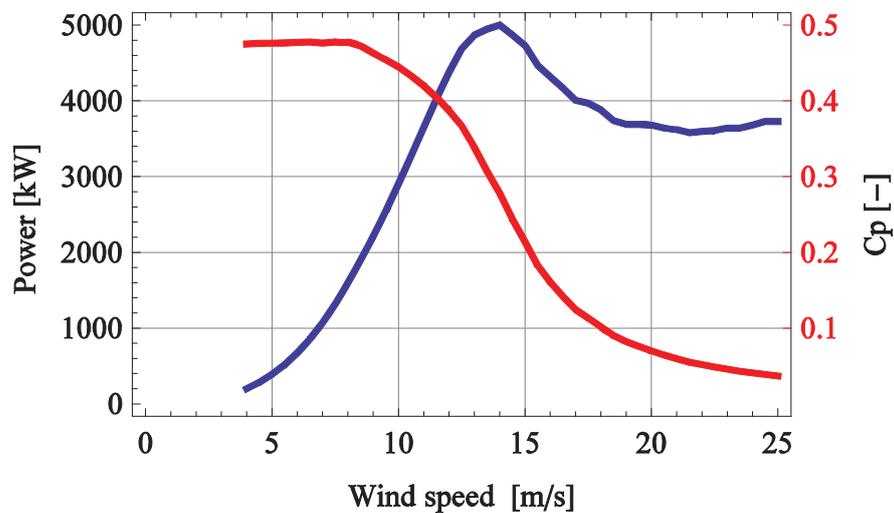
Paulsen US, Vita L, Madsen HA, Hattel J, Ritchie E, Leban KM, Berthelsen PA, Carstensen S *1st DeepWind 5 MW baseline design Energy Procedia 00 (2011) 000–000*

### Geometry

Rotor radius (R)	[m]	63.74
Rotor height (H)	[m]	129.56
Chord (c)	[m]	7.45
Solidity ( $\sigma = Nc/R$ )	[-]	0.23
Swept Area	[m <sup>2</sup> ]	10743

### Performance

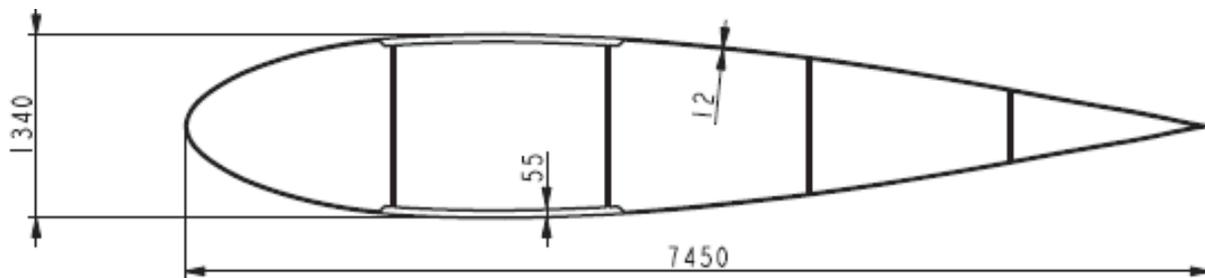
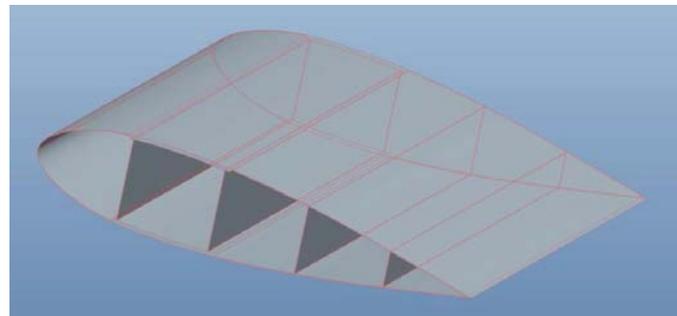
Rated power	[kW]	5000
Rated rotational speed	[rpm]	5.26
Rated wind speed	[m/s]	14
Cut in wind speed	[m/s]	5
Cut out wind speed	[m/s]	25



## DeepWind

# Results 1<sup>st</sup> BaseLine 5 MW Design Blades

- blade weight 154 Ton
- blade length 187 m
- Blade chord 7.45 m, constant over length
- All GRP
- NACA 0018 profile



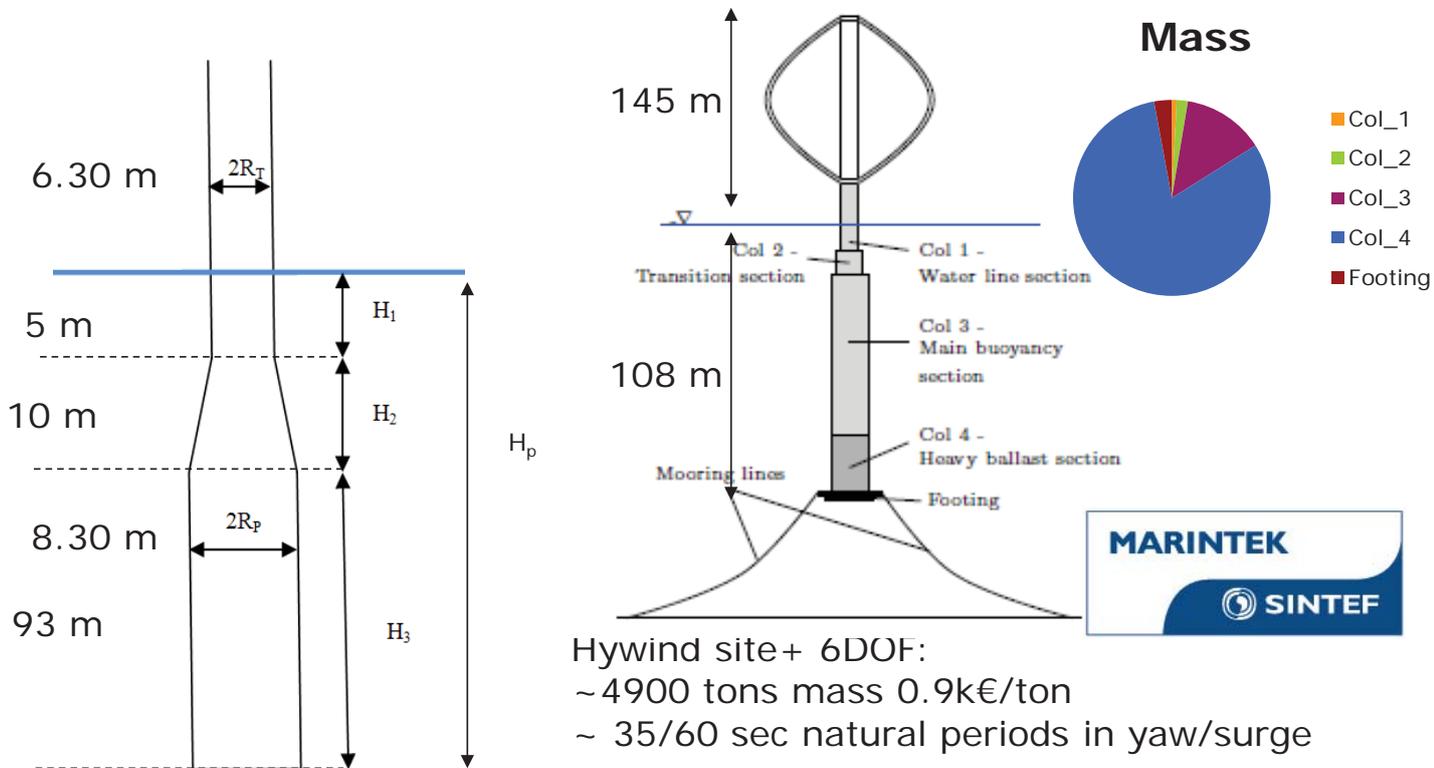
## DeepWind

# Results 1<sup>st</sup> BaseLine 5 MW Design Generator

- 5 MW mechanical power at estimated 5.26 rpm and 9.1 MNm shaft torque renders a 400 pole 17.53 Hz design with a pole pitch of around 7.85cm
  - This corresponds to an air-gap diameter of around 10 m outer diameter of around 10.5 m, with a core length of around 1.4 m.
  - Mass of Copper, Iron and permanent magnet materials of around 90 metric tons
  - Design fits reasonable with the platform design

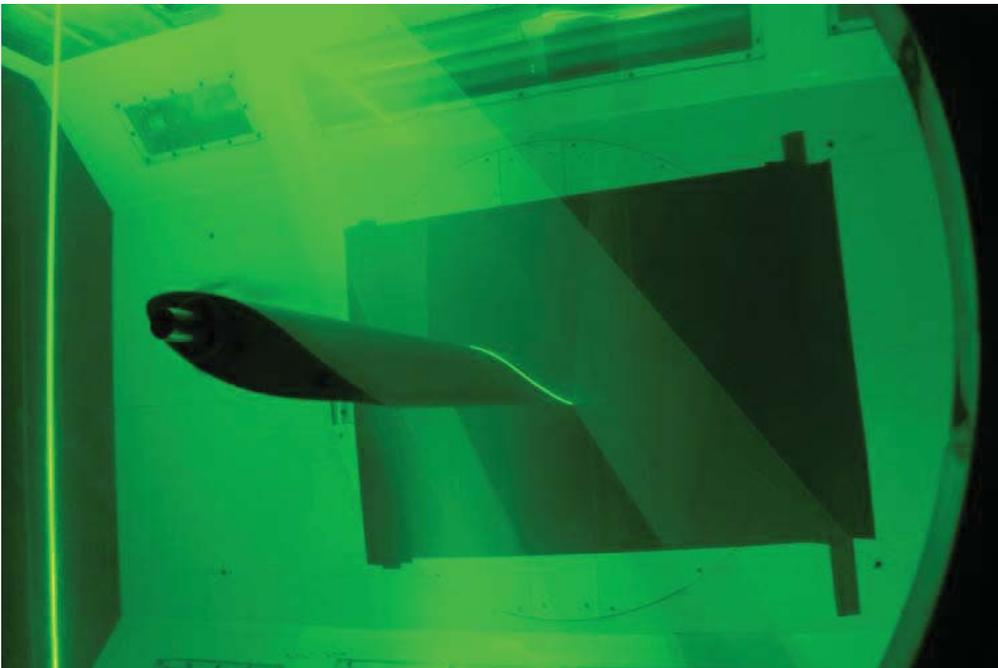
# DeepWind

## Results 1<sup>st</sup> BaseLine 5 MW Design Floater



□ Petter Andreas Berthelsen, Ivar Fylling, Luca Vita, Uwe Schmidt Paulsen CONCEPTUAL DESIGN OF A FLOATING SUPPORT STRUCTURE AND MOORING SYSTEM FOR A VERTICAL AXIS WIND TURBINE Proceedings of the ASME 2012 31st International Conference on Ocean, Offshore and Arctic Engineering OMAE2012 June 10–15, 2012, Rio de Janeiro, Brazil

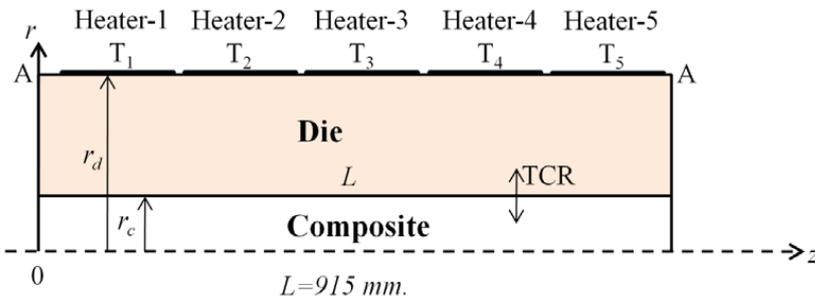
# DeepWind Results LTT Windtunnel tests (July 2012)



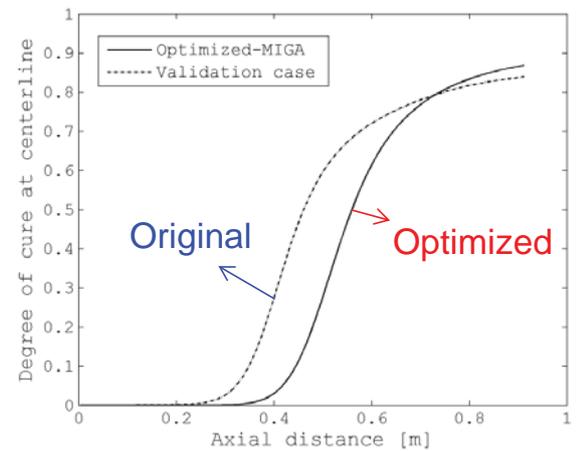
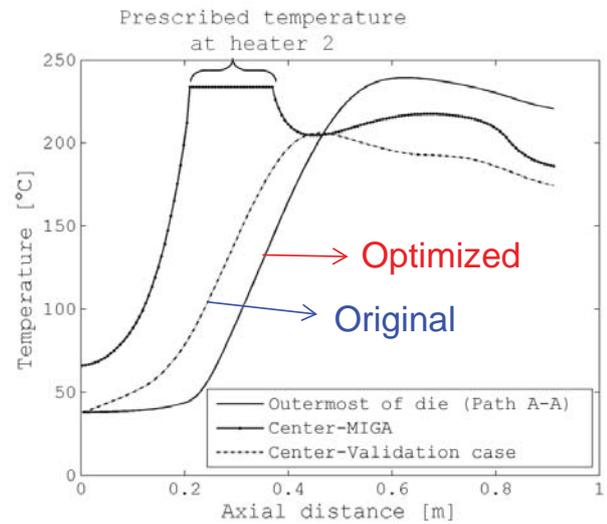
Analysis of advanced airfoils developed for VAWTs

**Studies already done:**

- Process optimization studies by using gradient based and/or genetic algorithms
  - Optimal heater configuration
  - Increase productivity i.e. increase pulling speed while satisfying the desired cure degree



Baran I, Tutum CC, Hattel JH. *App Compos Mat.* 2012. DOI: 10.1007/s10443-012-9278-3.



# DeepWind

## Results Physical Model Experiments

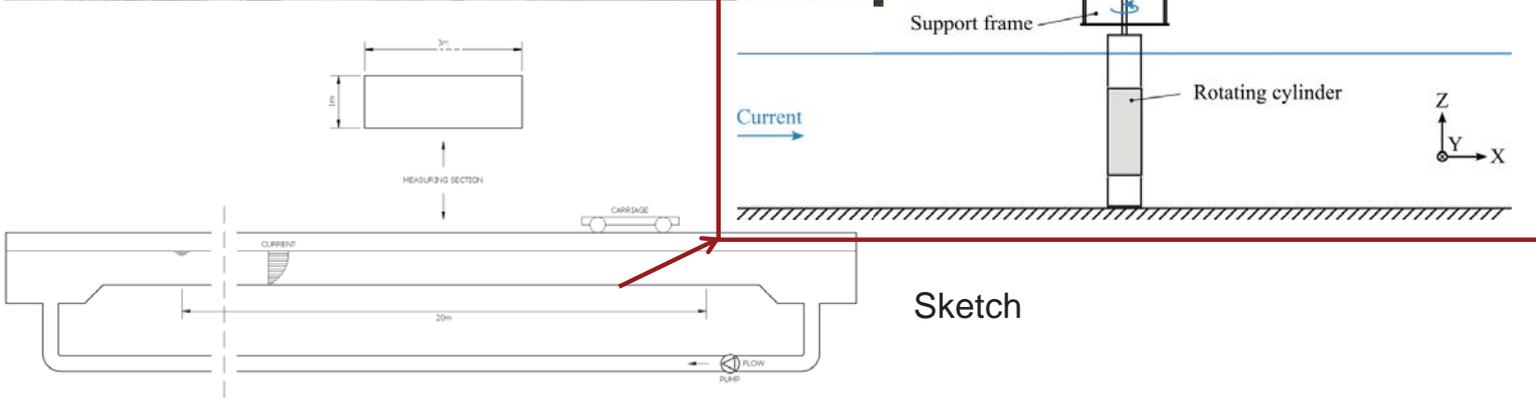


Picture



Carstensen et al *Lift of a Rotating Circular Cylinder in Unsteady Flows* ISOPE June 2012

Close-up (sketch)



Sketch

# DeepWind

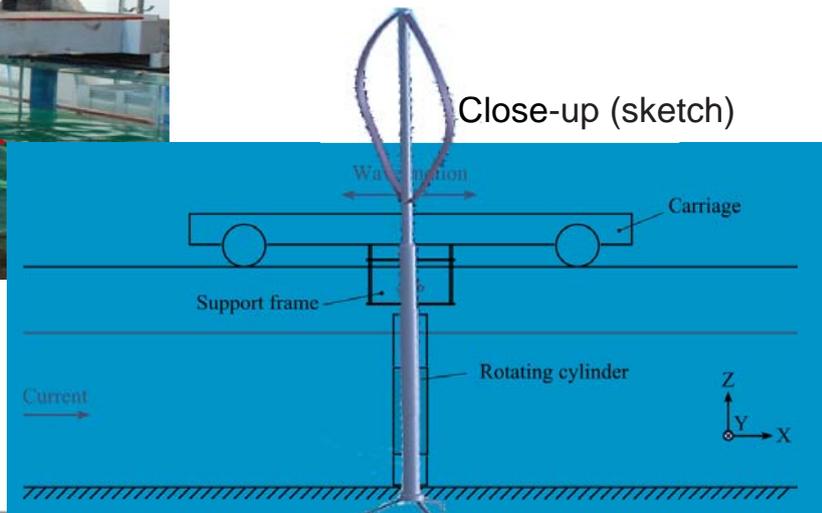
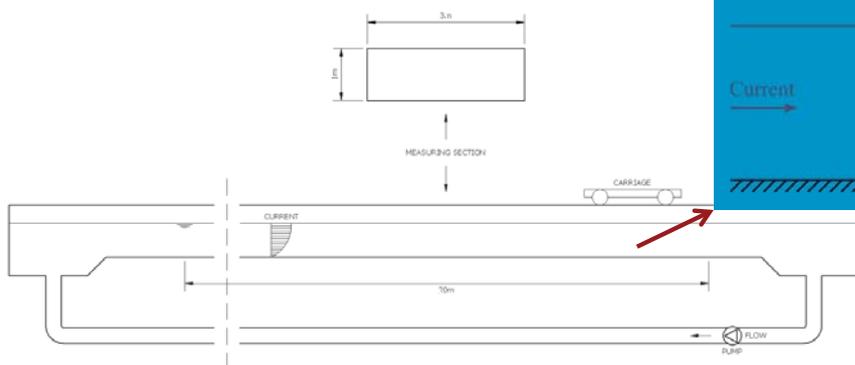
## Results Physical Model Experiments



Picture

$$U = U_m \sin(2\pi ft + \varphi) + U_c$$

*Carstensen et al Lift of a Rotating Circular Cylinder in Unsteady Flows ISOPE June 2012*



Close-up (sketch)

Sketch

DeepWind-an Innovative Floating Offshore Wind Turbine concept

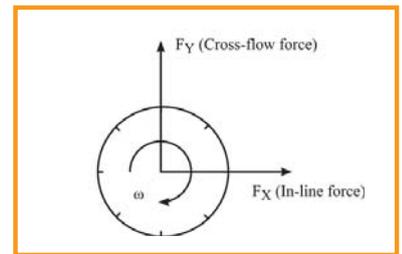
## DeepWind

# Results Physical Model Experiments Forces

(a) Morison formulation, waves and current:

Sketch

$$F_X = \frac{1}{2} \rho C_D D U |U| + \rho C_m A \frac{dU}{dt} \quad F_Y = 0$$



(b) Cylinder rotating in steady current:

$$F_X = \frac{1}{2} \rho C_D D U |U| \quad F_Y = \frac{1}{2} \rho C_L D U |U|$$

(c) Cylinder rotating in unsteady flow (Ideal Fluid):

$$F_X = \rho C_m A \frac{dU}{dt} \quad F_Y = \rho \Gamma U = \rho 2A \omega U = \rho C_r A \omega U$$

# DeepWind

## Results Physical Model Experiments Oscillatory Lift Force

*KC small: (1 < KC < 8)*

$$KC = \frac{2\pi a}{D} = \frac{U_m T}{D}$$

$$F_Y = \rho C_{r'} A \omega U + \rho C_{mY} A \frac{dU}{dt}$$

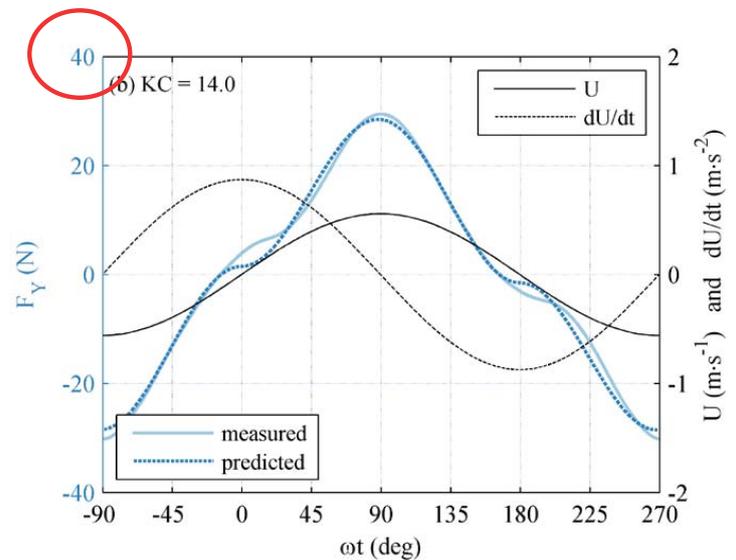
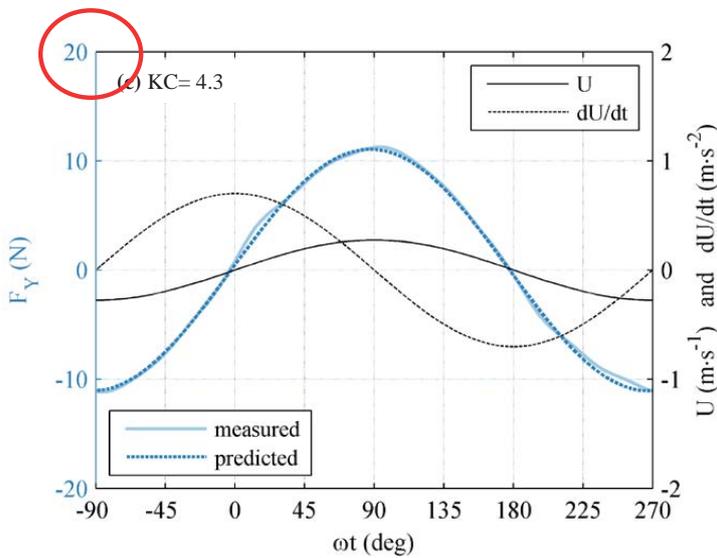
$$Re = \frac{a U_m}{\nu} = \frac{T U_m^2}{2\pi \nu}$$

$$T \cong C \sqrt{\frac{H}{g}} = C \sqrt{2ae^{-2\pi z/L}}$$

Carstensen et al *Lift of a Rotating Circular Cylinder in Unsteady Flows* ISOPE June 2012

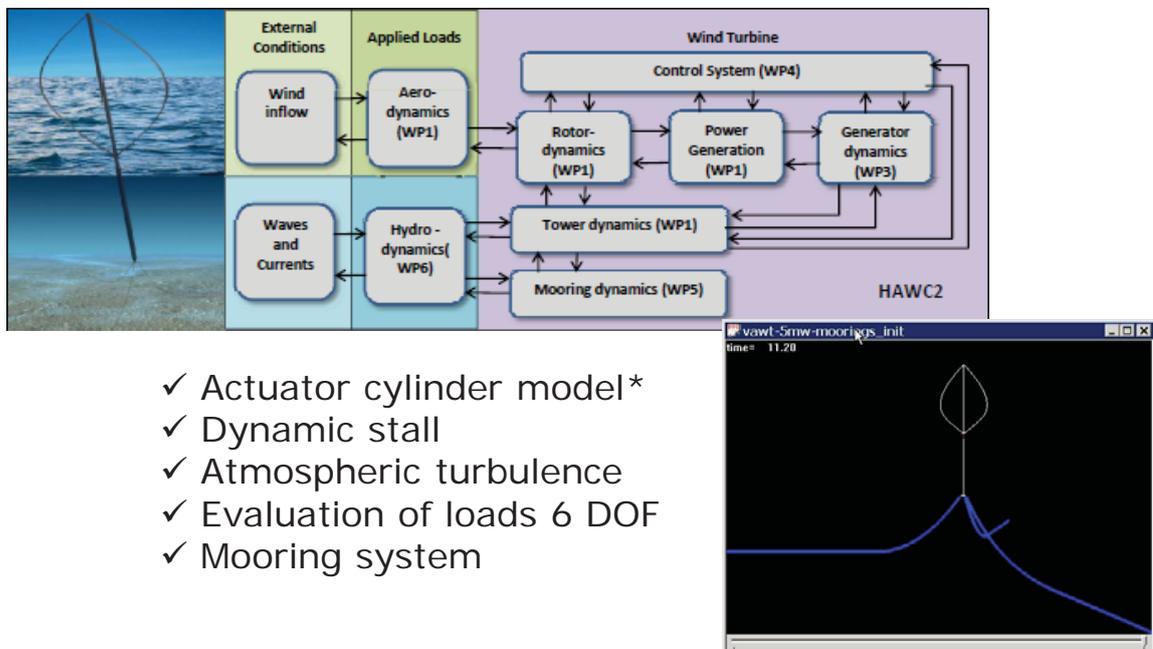
*KC large: (12 < KC < 24)*

$$F_Y = \frac{1}{2} \rho C_L D U |U| + \rho C_{mY} A \frac{dU}{dt}$$



# DeepWind

## Results 2nd Design Assumptions



- ✓ Actuator cylinder model\*
- ✓ Dynamic stall
- ✓ Atmospheric turbulence
- ✓ Evaluation of loads 6 DOF
- ✓ Mooring system

*\*Madsen HA, Larsen T, Paulsen US Adoption of the aeroelastic code HAWC2 for vertical axis turbines using the actuator cylinder flow model 51<sup>st</sup> AIAA conference Dallas Texas(USA) Jan 2013*

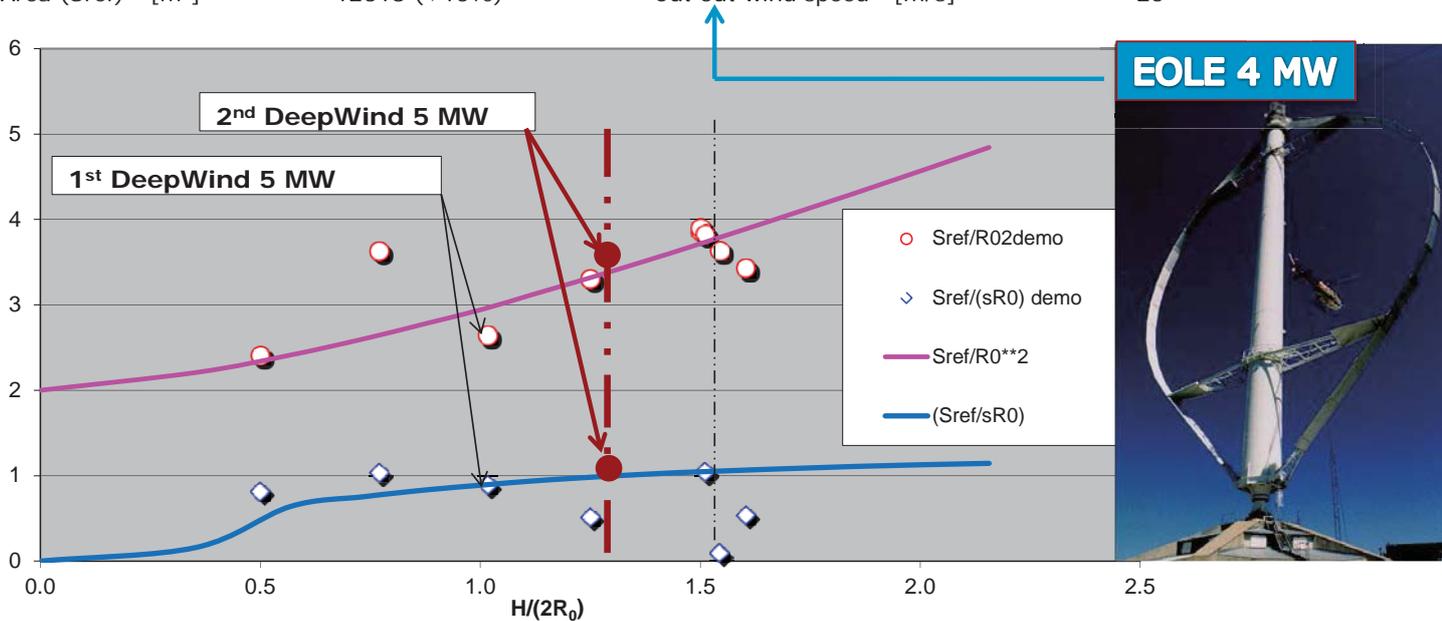
# DeepWind

## Results 2<sup>nd</sup> iteration 5 MW Design Rotor

### Geometry

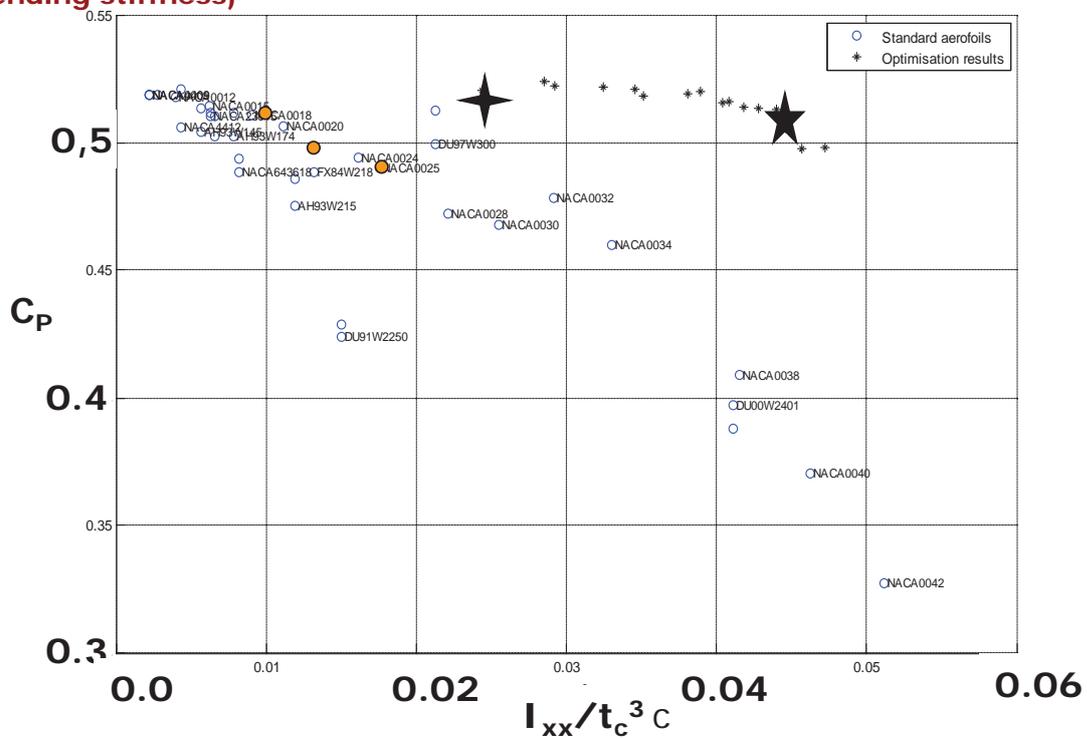
### Performance

Rotor radius (R0) [m]	58.5 (-8%)	Rated power [kW]	6200 (+24%)
Rotor Height (H) [m]	143 (+10.4%)	Rated rotational speed [rpm]	5.63 (+ 7%)
H/(2R0) [-]	1.222 (+22%)	Rated wind speed [m/s]	14
Solidity ( $\sigma = Nc/R0$ ) [-]	0.15 (-33%)	Cut in wind speed [m/s]	5
Swept Area (Sref) [m <sup>2</sup> ]	12318 (+15%)	Cut out wind speed [m/s]	25



# Results $C_p$ vs dimensionless flapwise Inertia

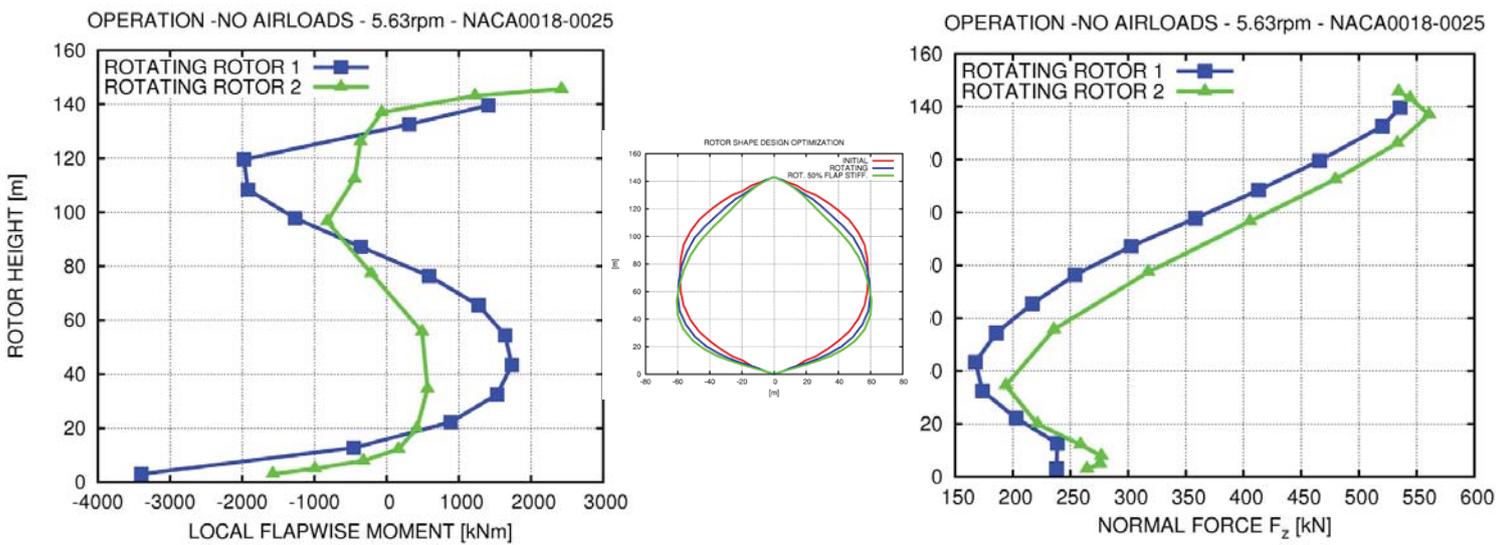
(bending stiffness)



©DeepWind, TUDelft

# DeepWind

## Results case-2+ 1iteration



□ Uwe Schmidt Paulsen, Helge Aagård Madsen, Ismet Baran, Per Nielsen, Jesper Hattel *Design Optimization of a 5 MW Floating Offshore Vertical Axis Wind Turbine* presented at the Deep Sea Offshore Conference Trondheim (NO) 2013

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## DeepWind

# Conclusion

- ✓ Established a full design tool and verification code integrating VAWT concept
  - ✓ Established a full model for blade pultrusion and preparation of advanced thick airfoils of laminar flow family with smaller  $CD_0$  and good  $C_p$
  - ✓ Design tool for PMG subsea generators; 1<sup>st</sup> design of a 5 MW generator
  - ✓ Design tool and verification tool for VAWT controls
  - ✓ 1<sup>st</sup> Floater for 5 MW design
  - ✓ Verified Fluid dynamics for rotating cylinders
  - ✓ Iteration from a 1<sup>st</sup> 5 MW floating concept to a 2<sup>nd</sup> iteration towards a light weight 5MW rotor with low bending moment
- 
- Continuation of iterations for improved design and for Cost analysis
  - 1<sup>st</sup> campaign of Demonstrator tests conducted
  - Next tests to be carried out in Ocean lab

## DeepWind Conclusion



# Thank You for Your Attention

# Q

Thanks to EU FP7 and the members of DeepWind consortium:  
DTU(DK), AAU(DK), TUDELFT(NL), TUTRENTO(I), DHI(DK),  
SINTEF(N), MARINTEK(N), MARIN(NL), NREL(USA), STATOIL(N),  
VESTAS(DK) and NENUPHAR(F).